

A solid model assembly of a 24 kHz Bone Tip and piezoelectric transducer is shown in Fig. 2. The PZT (Lead Zirconate Titanate) stack transducer, internal stepped horns, and surgical tip are accurately represented in the solid model. Measures (shown with balance symbols) are defined for stack and horn distal end acceleration and displacement. The maximum stress in the surgical tip is monitored. Ultimately, the nodal force is applied uniformly in proximity to the relative node location determined from modal analysis of resonant modes of the model. Surgical horn geometries have become quite complex, as shown in this solid model and picture of a Bone Tip device, with a stepped horn of area ratio gain, Gaussian horn, short straight section, growing exponential, and inverses cone. A helical abrasive pyramidal structure is machined in the inverse cone.

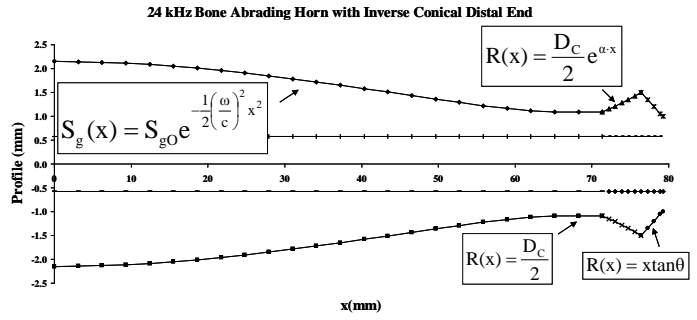


Figure 3. Physical-mathematical Model. Equations are still written for the simplified horn profiles; however, complex contours with asymmetric geometries and structure that cannot be practically modeled with physical equations make 3-D Finite Element Method essential.

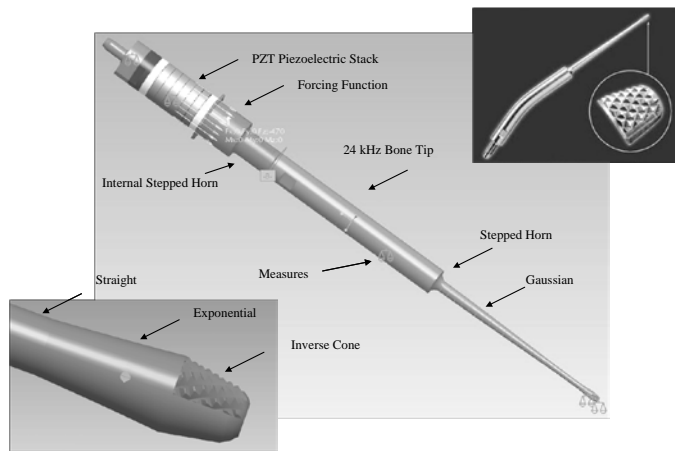


Figure 2. FEM Solid Model. An assembly of a 24 kHz Bone Tip and piezoelectric transducer is shown, along with a picture of the surgical tip. Surgical tip includes a stepped horn, Gaussian, straight section, exponential, and abrasive surface on an inverse cone.

B. Modeling of Ultrasonic Transducers and Horns

The 1-D (one-dimensional) physical-mathematical models or equations are still written for the simplified horn geometries, Fig. 3, to provide understanding of parametric adjustment of frequency, gain, and stress; however, complex contours with asymmetric geometries and structure that cannot be practically modeled with physical equations make 3-D FEM (Finite Element Method) essential. The Bone Tip provides examples of the benefit of FEM: as the wall thickness of the horn is increased, frequency increases due to increasing stiffness of a greater wall thickness cylinder, although mass is being added to the horn, and this is not apparent from the 1-D physical-mathematical model. The pyramidal abrasive structure reduces frequency by about 120 Hz due to the reduction in mass near the antinode, or distal end of the horn. Writing physical equations for this change in geometry and its influence on a resonant frequency dominated by the Gaussian horn is impractical.

Herein, a Pro/Mechanica analysis method published by the authors of this paper is extended from magnetostrictive to piezoelectric transducers, and reference is provided to this earlier publication, which has more discussion of the basic analysis approaches and possible output of results [7]. Pro/Mechanica is a product of Parametric Technology Corporation, Needham MA. Publications provide additional basis for understanding transducer-horn models and mechanics of wave motion [16-20], examples of benefits of FEM [21-22], and indicate another developer used Pro/Mechanica for analyzing ultrasonic horns for welding [23].

II. APPROACH TO FINITE ELEMENT METHOD FOR SURGICAL HORNS, IN BRIEF

Full model approach to Design Frequency Analysis utilizes a forcing function with damping and no artificial constraints. A modal analysis is first executed with the model to determine resonant modes. The force employed in the Design Frequency Analysis (e.g., about 2090 N) is that level provided by transducer stack at 100% amplitude. It is the nodal force and stress at the node that supports strain and elongation necessary to surgical tip stroke. The damping used with the forcing function in Design Frequency Analysis is established to provide the controlled magnitude of stack displacement or known surgical tip displacement. The forcing function analysis of the full model enables the full motion of the components and assembly to be evaluated independent of artificial constraints. Artificial constraints can limit real motion, cause greater frequency resonance due to artificial stiffening of the geometry, and lead to somewhat greater magnitude stress predictions. Artificial surface constraints in a half-model modal analysis can be useful in a wideband analysis (e.g., 5 kHz to 50 kHz) to show the dominant modes associated with the allowed frequencies of the geometry. Full model approach with no artificial constraints yields many modes to be evaluated to determine which are providing predominantly longitudinal motion.

A half model approach with symmetry constraints or full model approach with no artificial constraints will typically be performed over a narrow band about resonance, which may be iteratively reduced to speed processing. The half model

approach with symmetry constraints is quite rapid over a narrow band (e.g., 10 minutes of processing time), and it supports iterative design. The half model or sectioned full model also afford unambiguous review of internal stresses of the horn or other components of the assembly.

III. FINITE ELEMENT METHOD OUTPUT AND SIMULATIONS

A. Horn Stroke, Stress, and Vibration Simulation

Stroke levels at pertinent locations and relative nodes (local minimum in motion) and antinodes (local maximum in motion) in the horns are shown in Fig. 4. In the present design, the standing wave is orchestrated similarly for the new and baseline horns and the horn stroke is made approximately the same magnitude; consequently, a datum point at the nut of the stack could also be normalized for stroke.

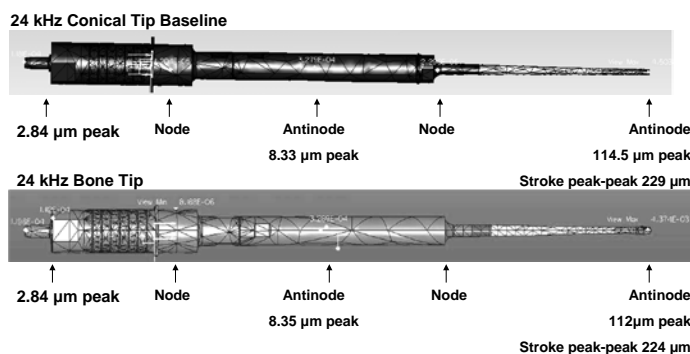


Figure 4. Mechanics Design Frequency Analysis. Stroke levels at pertinent locations and relative nodes and antinodes.

Maximum surgical tip stress locations are exhibited in Fig. 5 for the horn based on a given nodal force in the transducer and achieved stroke. There are many other analysis outputs that are not displayed well in a technical paper; for example, the modal analysis fringe plots show simulated motion dynamically and at selected magnification, so the full motion and any errant modes, such as flexure or ripple are magnified. The Master Interval Analysis enables displacement, stress, strain, velocity, and other selected parameters to be displayed at resonance and queried graphically as a function of position.

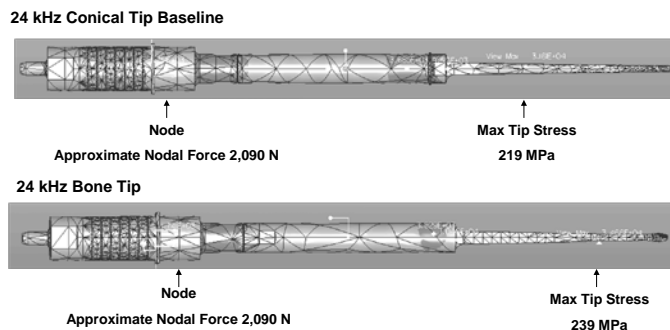


Figure 5. Mechanics Stress Fringe Plots. Maximum surgical tip stress locations are shown for the horn based on a given nodal force.

B. Nodal Force

Mechanics analysis enables calculation of nodal force given known cross-sectional area multiplied by queried stress. It should be noted that damping percentage can be adjusted to establish a known stroke at a datum or distal end of the surgical tip. If the nodal force were initially unknown, a starting value could be entered and the real nodal force determined based on queried stress at a node with the horn at full stroke. Establishing a basis value of nodal force, as well as confirming material properties, with a baseline device is helpful or an iterative approach is necessary. Fig. 6 indicates a potential problem with interrogation of nodal stress or nodal force per unit area at a relative node. High mechanical gain piezoelectric transducers can have large stress concentration due to stepped horns. A query in the stress concentration region could confound determination of nodal force. If a half-wavelength horn extender exists, such that stress is made more planar in the thin-wall, the nodal force could be approximated along the horn. If the relative node is in the radius of curvature of a stepped horn, for example at the end of the extender as shown, the query of nodal stress can be ambiguous due to steep gradients in stress and cross-sectional area.

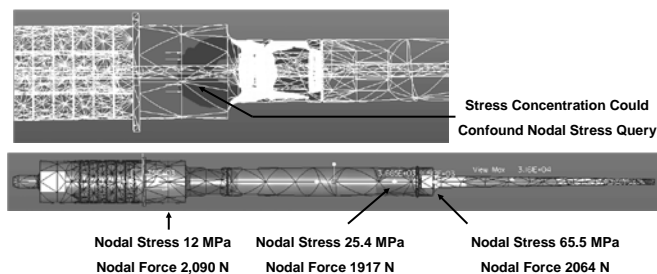


Figure 6. Calculation of Nodal Force. A query in the stress concentration region could confound determination of nodal force. If a half-wavelength horn extender exists, the nodal force could be approximated along the horn. Query of nodal stress can be ambiguous due to steep gradients in stress and cross-sectional area in a radius of curvature.

IV. SUMMARY OF RESULTS AND DISCUSSION

A. Finite Element Results Tabulated

A summary of Mechanics FEM results is provided in Table 1 for the representative 24 kHz Bone Tip and a 24 kHz Conical Tip baseline device. The results shown are from the Design Frequency Analysis using a full model with a forcing function. Transducer stack peak displacement is normalized at the controlled magnitude about 2.84 μm, which is governed by the closed-loop feedback of the ultrasonic controller. All stress analysis and values displayed herein are based on the von-Mises Stress Criterion. Maximum horn stress yielded by the forcing function analysis of the 24 kHz Bone Tip and 24 kHz Conical Tip baseline device are maintained at approximately 1/3 the yield strength of the titanium. Stress concentrations are also of concern due to potential impact on cyclic fatigue life of the metal. The new horn design results in similar stress to the baseline device. Of course, horns of this design have been successfully employed in surgical applications at similar

allowed stress levels for many years. Stroke of the 24 kHz Bone Tip was designed to be about 229 μm p-p, a value similar to the baseline, known to be adequate for precision bone fragmentation. This was favorable to ensuring similar stress levels and operation with the existing analog controller.

Table 1. Summary of Finite Element Analysis

	24 kHz Bone Tip	24 kHz Conical Tip
Stack Displacement peak (μm)	2.84	2.84
Stack Acceleration (m/s^2)	68.6E+3	68.6E+3
Horn von Mises Stress (MPa)	237	219
Horn Stroke peak-peak (μm)	224	229
Horn Acceleration (m/s^2)	2.69E+06	2.74E+06
Resonant Frequency (Hz)	24,740	24,610
Input Forcing Function (N)	2,091	2,091
Damping (%)	0.01424	0.0848

B. Experimental Electromechanical Data

In support of the FEM results, it is important to note that the actual measured strokes of the prototype and baseline horns were in reasonable agreement with Mechanics analysis, as shown in Table 2. Horn stroke is routinely measured optically with a high magnification microscope, CCD camera, monitor, and calibrated "Video Scaler". Data are shown for a sample set of 15 prototype horns and production baseline data. The agreement to the FEM stroke results for the full model and forcing function was within 5 μm for both the new and baseline horns. This represents less than a 2.5% error (e.g. 5 μm in 224 μm p-p) for the new geometry and baseline horn of this study. By incorporating the referenced work by the authors of this paper, 4 other models have shown 2.5 % to 6.8% error (8 μm error in 117 μm p-p) depending on the complexity of the model. Finite Element Method supports horn design, and the analysis coupled with extensive verification and life testing has supported surgical tip development.

Table 2. Electromechanical Data from Fabricated Horns

		Voltage	Current	Power	Frequency	Stroke
		(V_{RMS})	(mA_{RMS})	(Watts)	(kHz)	(μm)
24 kHz Bone Tip	Average	153	288	17	23.87	229
	StdDev	21	21	0.6	0.02	3.9
24 kHz Conical Tip	Average	285	477	28	23.76	229
	StdDev	11	15	1.0	0.05	3.9

C. Resonant Frequency Determined

Modeled resonant frequency of 24,610 Hz for the baseline Conical tip is about 850 Hz greater than the measured electromechanical data of fabricated devices (23,760 Hz), and this frequency shift has been common to all transducer-horn systems analyzed to date. Compliance in the joints is greater than modeled, elastic modulus of titanium material is likely lower than specified, and comparison electromechanical data are acquired at resonance mounted in the handpiece under full stroke quiescent operation, where heating reduces elastic modulus. The new Bone Tip was designed to be closer to an ideal resonance at 24 kHz. Analysis yielded 24,740 Hz, and given the 850 Hz shift expected, 23,890 Hz was predicted for the new horn. The actual electromechanical data averaged 23,870 Hz, or about a 20 Hz error, and clearly within the range of variance of the material properties and measurements. The Finite Element Method provides a practical means of fabricating tips about resonance in first-pass of machining.

D. Surgical Bone Tip Employed in Cadaver-Section

The Bone Tip is shown in Fig. 7 fragmenting and aspirating hard skull bone as the neurosurgeon is comparing removal with other instruments in a cadaver-section. Precision bone removal and thermal management is afforded with the newly released surgical tip.

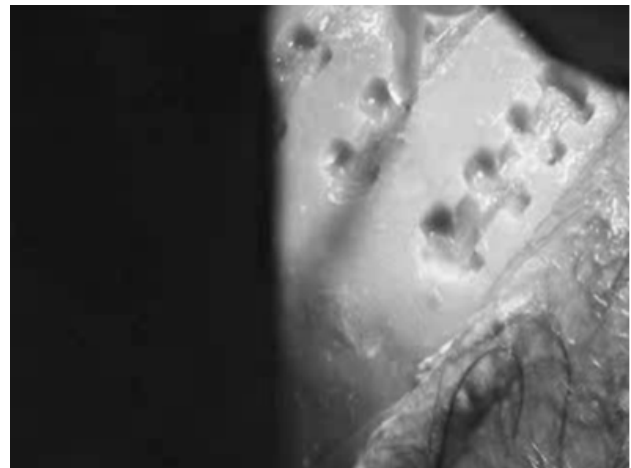


Figure 7. Fragmenting and Aspirating Hard Skull Bone. Precision bone removal and thermal management are afforded with the newly released surgical tip.

V. CONCLUSIONS

Surgical tips with complex contours, asymmetric geometries, and structure that cannot be practically modeled with physical-mathematical equations make 3-D Finite Element Methods essential. Agreement of Finite Element Method results and electromechanical data were within 2.5% error (e.g. 5 μm in 224 μm p-p) for the new geometry and baseline horn of this study. Finite Element Method supports horn design, and analysis afforded coupled with extensive verification and life testing have supported surgical tip development. The Finite Element Method provides a practical means of fabricating tips about design resonant frequency in first-pass of machining. Precision bone removal and thermal management are afforded with the newly released surgical tip.

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